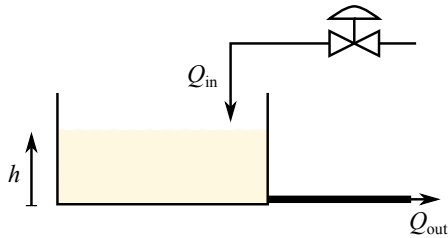


Process description: diff. equation, linearization



Aim: height control.

What is the dependance between flows and liquid level?

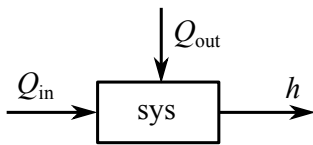
Mass conservation law

$$m_{acc} = m_{in} - m_{out}, \text{ if } \rho = \text{const}$$

$$\frac{dV}{dt} = Q_{in} - Q_{out}$$

$$V = A \cdot h$$

1 Differential equation



$$h = \frac{1}{A} \int (Q_{in} - Q_{out}) dt \quad (1)$$

What is the Q_{out} ?

It depends on the pressure on the bottom

$$P = \frac{F}{A} = \frac{m \cdot g}{A} = \frac{\rho \cdot V \cdot g}{A} = \frac{\rho \cdot h \cdot A \cdot g}{A} = \rho h g$$

In that case

$$Q_{out} \equiv n^* \sqrt{P} \quad \Rightarrow \quad Q_{out} = n \sqrt{h},$$

where n^* - coeff. dependant on the properties of the liquid and valve.

Thus, flow rate through a restriction such as discharge valve is

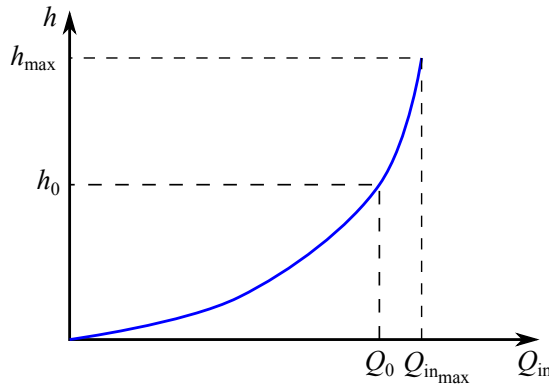
$$\frac{dh}{dt} = \frac{1}{A} (Q_{in} - n \sqrt{h}) = \frac{1}{A} Q_{in} - \frac{n}{A} \sqrt{h}$$

$$\frac{dh}{dt} = \frac{1}{A} Q_{in} - \frac{n}{A} \sqrt{h} \quad \text{- Process model} \quad (2)$$

2 Linearization

a) Steady-state $\frac{df}{dt} = 0$

$$\begin{aligned} \dot{h} + \frac{n}{A}\sqrt{h} - \frac{1}{A}Q_{in} &= 0 \\ \dot{h} &= k_1 Q_{in} - k_2 \sqrt{h} \end{aligned} \quad \left| \quad \begin{aligned} F(h, \dot{h}, Q_{in}) &= 0 \\ h &= f(h, Q_{in}) \\ k_1 &= \frac{1}{A} \\ k_2 &= \frac{n}{A} \end{aligned} \right.$$

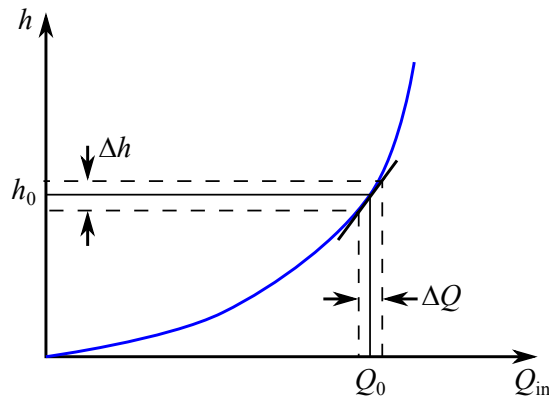


$$\begin{aligned} \dot{h} = 0 &\Rightarrow Q_{in} = Q_{out} \\ k_1 Q_{in} &= k_2 Q_{out} \\ Q_{in} &\equiv \sqrt{h} \\ Q_{in}^2 &= h \end{aligned}$$

$$h = f(h, Q_{in})$$

$$f(Q_{in_0}, h_0) = 0 \quad \text{steady state}$$

At a working point h_0 we want to linearize the function



$$\begin{aligned} Q_{in} &= Q_{in_0} + \Delta Q_{in} \\ h &= h_0 + \Delta h \end{aligned}$$

$$f(Q_{in_0} + \Delta Q_{in}, h_0 + \Delta h_0) = \overbrace{f(Q_{in_0}, h_0)}^0 + \left. \frac{\partial f}{\partial Q_{in}} \right|_{\substack{Q_{in_0} \\ h_0}} \cdot \Delta Q_{in} + \left. \frac{\partial f}{\partial h} \right|_{\substack{Q_{in_0} \\ h_0}} \cdot \Delta h$$

$$\Delta \dot{h} = k_1 \Delta Q_{in} + k_2 \frac{1}{2} \frac{1}{\sqrt{h_0}} \cdot \Delta h = k_1 \Delta Q_{in} - \frac{k_2}{2\sqrt{h_0}} \cdot \Delta h \quad \text{linearized equation} \quad (3)$$

3 Laplace domain representation $W(s) = \frac{K}{1 + T \cdot s}$:

$$\begin{aligned} \frac{d\Delta h}{dt} + \frac{k_2}{2\sqrt{h_0}} \cdot \Delta h &= k_1 \Delta Q_{in} \\ y + T \cdot s \frac{dy}{dt} &= Ku \end{aligned} \quad \Rightarrow \quad \Delta h + \underbrace{\frac{2\sqrt{h_0}}{k_2}}_{T \cdot s} \frac{d\Delta h}{dt} = \underbrace{\frac{2k_1\sqrt{h_0}}{k_2}}_K \Delta Q_{in}$$

$$W = \frac{\frac{2k_1\sqrt{h_0}}{k_2}}{1 + \frac{2\sqrt{h_0}}{k_2}s} = \frac{\frac{2\frac{1}{A}\sqrt{h_0}}{\frac{n}{A}}}{1 + \frac{2\sqrt{h_0}}{\frac{n}{A}}s} = \frac{\frac{2\sqrt{h_0}}{n}}{1 + \frac{2A\sqrt{h_0}}{n}s} \quad (4)$$